

Rolling resistance

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Rolling resistance, sometimes called **rolling friction** or **rolling drag**, is the force resisting the motion when a body (such as a ball, tire, or wheel) rolls on a surface. It is mainly caused by non-elastic effects; that is, not all the energy needed for deformation (or movement) of the wheel, roadbed, etc. is recovered when the pressure is removed. Two forms of this are hysteresis losses (see below), and permanent (plastic) deformation of the object or the surface (e.g. soil). Another cause of rolling resistance lies in the slippage between the wheel and the surface, which dissipates energy. Note that only the last of these effects involves friction, therefore the name "rolling friction" is to an extent a misnomer.

In analogy with sliding friction, rolling resistance is often expressed as a coefficient times the normal force. This coefficient of rolling resistance is generally much smaller than the coefficient of sliding friction.^[1]

Any coasting wheeled vehicle will gradually slow down due to rolling resistance including that of the bearings, but a train car with steel wheels running on steel rails will roll farther than a bus of the same mass with rubber tires running on tarmac. Factors that contribute to rolling resistance are the (amount of) deformation of the wheels, the deformation of the roadbed surface, and movement below the surface. Additional contributing factors include wheel diameter, speed,^[2] load on wheel, surface adhesion, sliding, and relative micro-sliding between the surfaces of contact. The losses due to hysteresis also depend strongly on the material properties of the wheel or tire and the surface. For example, a rubber tire will have higher rolling resistance on a paved road than a steel railroad wheel on a steel rail. Also, sand on the ground will give more rolling resistance than concrete.

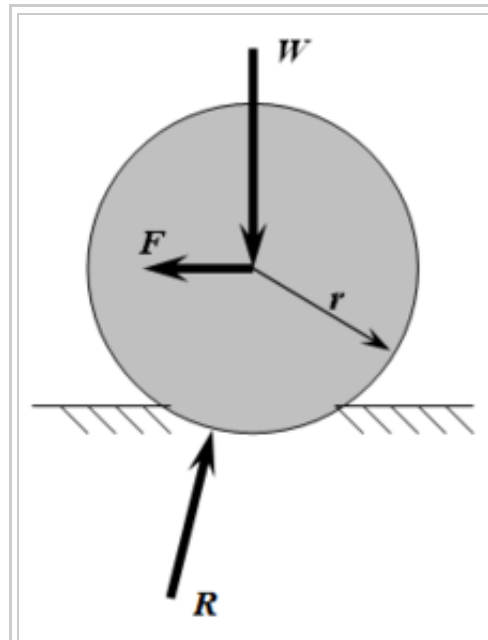


Figure 1 Hard wheel rolling on and deforming a soft surface, resulting in the reaction force R from the surface having a component that opposes the motion. (W is some vertical load on the axle, F is some towing force applied to the axle, r is the wheel radius, and both friction with the ground and friction at the axle are assumed to be negligible and so are not shown. The wheel is rolling to the left at constant speed.) Note that R is the resultant force from non-uniform pressure at the wheel-roadbed contact surface. This pressure is greater towards the front of the wheel due to hysteresis.

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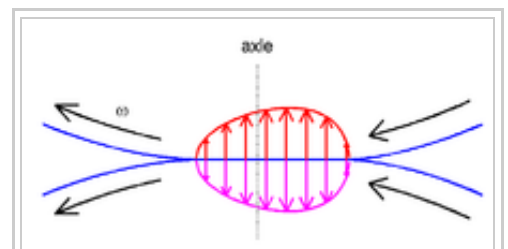
Primary cause

The primary cause of pneumatic tire rolling resistance is hysteresis:^[4]

A characteristic of a deformable material such that the energy of deformation is greater than the energy of recovery. The rubber compound in a tire exhibits hysteresis. As the tire rotates under the weight of the vehicle, it experiences repeated cycles of deformation and recovery, and it dissipates the hysteresis energy loss as heat. Hysteresis is the main cause of energy loss associated with rolling resistance and is attributed to the viscoelastic characteristics of the rubber.

-- National Academy of Sciences^[5]

This main principle is illustrated in the figure of the rolling cylinders. If two equal cylinders are pressed together then the contact surface is flat. In the absence of surface friction, contact stresses are normal (i.e. perpendicular) to the contact surface. Consider a particle that enters the contact area at the right side, travels through the contact patch and leaves at the left side. Initially its vertical deformation is increasing, which is resisted by the hysteresis effect. Therefore an additional pressure is generated to



Asymmetrical pressure distribution between rolling cylinders due to viscoelastic material behavior (rolling to the right).^[3]

avoid interpenetration of the two surfaces. Later its vertical deformation is decreasing. This is again resisted by the hysteresis effect. In this case this decreases the pressure that is needed to keep the two bodies separate.

The resulting pressure distribution is asymmetrical and is shifted to the right. The line of action of the (aggregate) vertical force no longer passes through the centers of the cylinders. This means that a moment occurs that tends to retard the rolling motion.

Materials that have a large hysteresis effect, such as rubber, which bounce back slowly, exhibit more rolling resistance than materials with a small hysteresis effect that bounce back more quickly and more completely, such as steel or silica. Low rolling resistance tires typically incorporate silica in place of carbon black in their tread compounds to reduce low-frequency hysteresis without compromising traction.^[6] Note that railroads also have hysteresis in the roadbed structure.^[7]

"Rolling resistance" has different definitions

In the broad sense, specific "rolling resistance" (for vehicles) is the force per unit vehicle weight required to move the vehicle on level ground at a constant slow speed where aerodynamic drag (air resistance) is insignificant and also where there are no traction (motor) forces or brakes applied. In other words the vehicle would be coasting if it were not for the force to maintain constant speed. An example of such usage for railroads is [3]

(<http://www.istc.illinois.edu/about/SeminarPresentations/20091118.pdf>). This broad sense includes wheel bearing resistance, the energy dissipated by vibration and oscillation of both the roadbed and the vehicle, and sliding of the wheel on the roadbed surface (pavement or a rail).

But there is an even broader sense which would include energy wasted by wheel slippage due to the torque applied from the engine. This includes the increased power required due to the increased velocity of the wheels where the tangential velocity of the driving wheel(s) becomes greater than the vehicle speed due to slippage. Since power is equal to force times velocity and the wheel velocity has increased, the power required has increased accordingly.

The pure "rolling resistance" for a train is that which happens due to deformation and possible minor sliding at the wheel-road contact.^[8] For a rubber tire, an analogous energy loss happens over the entire tire, but it is still called "rolling resistance". In the broad sense, "rolling resistance" includes wheel bearing resistance, energy loss by shaking both the roadbed (and the earth underneath) and the vehicle itself, and by sliding of the wheel, road/rail contact. Railroad textbooks seem to cover all these resistance forces but do not call their sum "rolling resistance" (broad sense) as is done in this article. They just sum up all the resistance forces (including aerodynamic drag) and call the sum basic train resistance (or the like).^[9]

Since railroad rolling resistance in the broad sense may be a few times larger than just the pure rolling resistance^[10] reported values may be in serious conflict since they may be based on different definitions of "rolling resistance". The train's engines must of course, provide the energy to overcome this broad-sense rolling resistance.

For highway motor vehicles, there is obviously some energy dissipating in the shaking the roadway and earth beneath, shaking of the vehicle itself, and sliding of the tires. But other than the additional power required due to torque and wheel bearing friction, non-pure rolling resistance doesn't seem to have been investigated, possibly because the "pure" rolling resistance of a rubber tire is several times higher than the neglected resistances.^[11]

Rolling resistance coefficient

The "rolling resistance coefficient", is defined by the following equation:^[5]

$$F = C_{rr} N$$

where

F is the rolling resistance force (shown in figure 1),

C_{rr} is the dimensionless **rolling resistance coefficient** or **coefficient of rolling friction** (**CRF**), and

N is the normal force, the force perpendicular to the surface on which the wheel is rolling.

C_{rr} is the force needed to push (or tow) a wheeled vehicle forward (at constant speed on the level with no air resistance) per unit force of weight. It is assumed that all wheels are the same and bear identical weight. Thus: $C_{rr} = 0.01$ means that it would only take 0.01 pounds to tow a vehicle weighing one pound. For a 1000 pound vehicle it would take 1000 times more tow force or 10 pounds. One could say that C_{rr} is in lb(tow-force)/lb(vehicle weight). Since this lb/lb is force divided by force, C_{rr} is dimensionless. Multiply it by 100 and you get the percent (%) of the weight of the vehicle required to maintain slow steady speed. C_{rr} is often multiplied by 1000 to get the parts per thousand which is the same as kilograms (kg force) per metric ton (tonne = 1000 kg)^[12] which is the same as pounds of resistance per 1000 pounds of load or Newtons/kilo-Newton, etc. For the US railroads, lb/ton has been traditionally used which is just $2000C_{rr}$. Thus they are all just measures of resistance per unit vehicle weight. While they are all "specific resistances" sometimes they are just called "resistance" although they are really a coefficient (ratio) or a multiple thereof. If using pounds or kilograms as force units, mass is equal to weight (in earth's gravity a kilogram a mass weighs a kilogram and exerts a kilogram of force) so one could claim that C_{rr} is also the force per unit mass in such units. The SI system would use N/tonne (N/T) which is $1000gC_{rr}$ and is force per unit mass, where g is the acceleration of gravity in SI units (meters per second square).^[13]

The above shows resistance proportional to C_{rr} but does not explicitly show any variation with speed, loads, torque, surface roughness, diameter, tire inflation/wear, etc. because C_{rr} itself varies with those factors. It might seem from the above definition of C_{rr} that the rolling resistance is directly proportional to vehicle weight but it is not.

Measurement

There are at least two popular models for calculating rolling resistance.

1. "Rolling resistance coefficient (RRC). The value of the rolling resistance force divided by the wheel load. The Society of Automotive Engineers (SAE) has developed test practices to measure the RRC of tires. These tests (SAE J1269 and SAE J2452) are usually performed on new tires. When measured by using these standard test practices, most new passenger tires have reported RRCs ranging from 0.007 to 0.014."^[5] In the case of bicycle tires, values of 0.0025 to 0.005 are achieved.^[14] These coefficients are measured on rollers, with power meters on road surfaces, or with coast-down tests. In the latter two cases, the effect of air resistance must be subtracted or the tests performed at very low speeds.

2. The coefficient of rolling resistance b , which has the dimension of length, is approximately (due to the small-angle approximation of $\cos(\theta) = 1$) equal to the value of the rolling resistance force times the radius of the wheel divided by the wheel load.^[2]
3. ISO 18164:2005 is used to test rolling resistance in Europe.

The results of these tests can be hard for the general public to obtain as manufacturers prefer to publicize "comfort" and "performance".

Physical formulas

The coefficient of rolling resistance for a slow rigid wheel on a perfectly elastic surface, not adjusted for velocity, can be calculated by^{[1][2]}

$$C_{rr} = \sqrt{z/d}$$

where

z is the sinkage depth

d is the diameter of the rigid wheel

Empirical formula for C_{rr} for cast iron mine car wheels on steel rails.^[15]

$$C_{rr} = 0.0048(18/D)^{\frac{1}{2}}(100/W)^{\frac{1}{4}}$$

where

D is the wheel diameter in in.

W is the load on the wheel in lbs.

As an alternative to using C_{rr} one can use b which is a different **rolling resistance coefficient** or **coefficient of rolling friction** with dimension of length, It is defined by the following formula:^[2]

$$F = \frac{Nb}{r}$$

where

F is the rolling resistance force (shown in figure 1),

r is the wheel radius,

b is the **rolling resistance coefficient** or **coefficient of rolling friction** with dimension of length, and

N is the normal force (equal to W , not R , as shown in figure 1).

The above equation, where resistance is inversely proportional to radius r . seems to be based on the discredited "Coulomb's law" (Neither Coulomb's inverse square law nor Coulomb's law of friction). See #Depends on diameter. Equating this equation with the force per the #Rolling resistance coefficient, and

solving for b , gives $b = C_{rr} \cdot r$. Therefore, if a source gives rolling resistance coefficient (C_{rr}) as a dimensionless coefficient, it can be converted to b , having units of length, by multiplying C_{rr} by wheel radius r .

Rolling resistance coefficient examples

Table of rolling resistance coefficient examples: [4] (<http://auto.howstuffworks.com/tire4.htm>)

C_{rr}	b	Description
0.0003 to 0.0004 ^[16]		"Pure rolling resistance" Railroad steel wheel on steel rail
0.0010 to 0.0024 ^{[17][18]}	0.5 mm ^[2]	Railroad steel wheel on steel rail. Passenger rail car about 0.0020 ^[19]
0.001 to 0.0015 ^[20]	0.1 mm ^[2]	Hardened steel ball bearings on steel
0.0019 to 0.0065 ^[21]		Mine car cast iron wheels on steel rail
0.0022 to 0.005 ^[22]		Production bicycle tires at 120 psi (8.3 bar) and 50 km/h (31 mph), measured on rollers
0.0025 ^[23]		Special Michelin solar car/eco-marathon tires
0.005		Dirty tram rails (standard) with straights and curves
0.0045 to 0.008 ^[24]		Large truck (Semi) tires
0.0055 ^[23]		Typical BMX bicycle tires used for solar cars
0.0062 to 0.015 ^[25]		Car tire measurements
0.010 to 0.015 ^[26]		Ordinary car tires on concrete
0.0385 to 0.073 ^[27]		Stage coach (19th century) on dirt road. Soft snow on road for worst case.
0.3 ^[26]		Ordinary car tires on sand

For example, in earth gravity, a car of 1000 kg on asphalt will need a force of around 100 newtons for rolling ($1000 \text{ kg} \times 9.81 \text{ m/s}^2 \times 0.01 = 98.1 \text{ N}$).

Depends on diameter

Stagecoaches and railroads (diameter)

According to Dupuit (1837), rolling resistance (of wheeled carriages with wooden wheels with iron tires) is approximately inversely proportional to the square root of wheel diameter.^[28] This rule has been experimentally verified for cast iron wheels (8" - 24" diameter) on steel rail^[29] and for 19th century carriage wheels.^[27] But there are other tests of carriage wheels that do not agree.^[27] Theory of a cylinder rolling on an elastic roadway also gives this same rule^[30] These contradict earlier (1785) tests by Coulomb of rolling wooden cylinders where Coulomb reported that rolling resistance was inversely proportional to the diameter of the wheel (known as "Coulomb's law").^[31] This disputed (or wrongly applied) -"Coulomb's law" is still found in handbooks, however.

Pneumatic tires (diameter)

For pneumatic tires on hard pavement, it is reported that the effect of diameter on rolling resistance is negligible (within a practical range of diameters).^{[32][33]}

Depends on applied torque

The driving torque T to overcome rolling resistance R_r and maintain steady speed on level ground (with no air resistance) can be calculated by:

$$T = \frac{V_s}{\Omega} R_r$$

where

V_s is the linear speed of the body (at the axle), and
 Ω its rotational speed.

It is noteworthy that V_s/Ω is usually not equal to the radius of the rolling body.^{[34][35][36]}

All wheels (torque)

"Applied torque" may either be driving torque applied by a motor (often through a transmission) or a braking torque applied by brakes(including regenerative braking). Such torques results in energy dissipation (above that due to the basic rolling resistance of a freely rolling, non-driven, non-braked wheel). This additional loss is in part due to the fact that there is some slipping of the wheel, and for pneumatic tires, there is more flexing of the sidewalls due to the torque. Slip is defined such that a 2% slip means that the circumferential speed of the driving wheel exceeds the speed of the vehicle by 2%.

A small percentage slip can result in a much larger percentage increase in rolling resistance. For example, for pneumatic tires, a 5% slip can translate into a 200% increase in rolling resistance.^[37] This is partly because the tractive force applied during this slip is many times greater than the rolling resistance force and thus much more power per unit velocity is being applied (recall power = force x velocity so that power per unit of velocity is just force). So just a small percentage increase in circumferential velocity due to slip can translate into a loss of traction power which may even exceed the power loss due to basic (ordinary) rolling resistance. For railroads, this effect may be even more pronounced due to the low rolling resistance of steel wheels.

Railroad steel wheels (torque)

In order to apply any traction to the wheels some slippage of the wheel is required.^[38] For Russian trains climbing up a grade, this slip is normally 1.5% to 2.5%.

Slip (also known as creep)is normally roughly directly proportional to tractive effort. An exception is if the tractive effort is so high that the wheel is close to substantial slipping (more than just a few percent as discussed above), then slip rapidly increases with tractive effort and is no longer linear. With a little higher applied tractive effort the wheel spins out of control and the adhesion drops resulting in the wheel spinning even faster. This is the type of slipping that is observable by eye—the slip of say 2% for traction is only observed by instruments. Such rapid slip may result in excessive wear or damage.

Pneumatic tires (torque)

Rolling resistance greatly increases with applied torque. At high torques, which apply a tangential force to the road of about half the weight of the vehicle, the rolling resistance may triple (a 200% increase).^[37] This is in part due to a slip of about 5%. See #All wheels for an explanation of why this is reasonable. The rolling resistance increase with applied torque is not linear, but increases at a faster rate as the torque becomes higher.

Depends on wheel load

Railroad steel wheels (load)

The #Rolling resistance coefficient, C_{rr} , significantly decreases as the weight of the rail car per wheel increases.^[39] For example, an empty Russian freight car had about twice the C_{rr} as loaded car ($C_{rr}=0.002$ vs. $C_{rr}=0.001$). This same "economy of scale" shows up in testing of mine rail cars.^[40] The theoretical C_{rr} for a rigid wheel rolling on an elastic roadbed shows C_{rr} inversely proportional to the square root of the load.^[30]

If C_{rr} is itself dependent on wheel load per an inverse square-root rule, then for an increase in load of 2% only a 1% increase in rolling resistance occurs.^[41]

Pneumatic tires (load)

For pneumatic tires, the direction of change in C_{rr} (#Rolling resistance coefficient) depends on whether or not tire inflation is increased with increasing load.^[42] It is reported that if inflation pressure is increased with load according to an (undefined) "schedule", then a 20% increase in load decreases C_{rr} by 3%. But if the inflation pressure is not changed, then a 20% increase in load results in a 4% increase in C_{rr} . Of course this will increase the rolling resistance by 20% due to the increase in load plus $1.2 \times 4\%$ due to the increase in C_{rr} resulting in a 24.8% increase in rolling resistance.

Depends on curvature of roadway

General

When a vehicle (motor vehicle or railroad train) goes around a curve, rolling resistance usually increases. If the curve is not banked so as to exactly counter the centrifugal force with an equal and opposing centripetal force due to the banking, then there will be a net unbalanced sideways force on the vehicle. This will result in increased rolling resistance. Banking is also known as "superelevation" or "cant" (not to be confused with rail cant of a rail). For railroads, this is called curve resistance but for roads it has (at least once) been called rolling resistance due to cornering.

Sound effects

Rolling friction generates sound (vibrational) energy, as mechanical energy is converted to this form of energy due to the friction. One of the most common examples of rolling friction is the movement of motor vehicle tires on a roadway, a process which generates sound as a by-product.^[43] The sound

generated by automobile and truck tires as they roll (especially noticeable at highway speeds) is mostly due to the percussion of the tire treads, and compression (and subsequent decompression) of air temporarily captured within the treads.^[44]

Factors that contribute in tires

Several factors affect the magnitude of rolling resistance a tire generates:

- As mentioned in the introduction: wheel radius, forward speed, surface adhesion, and relative micro-sliding.
- Material - different fillers and polymers in tire composition can improve traction while reducing hysteresis. The replacement of some carbon black with higher-priced silica–silane is one common way of reducing rolling resistance.^[5] The use of exotic materials including nano-clay has been shown to reduce rolling resistance in high performance rubber tires.^[45] Solvents may also be used to swell solid tires, decreasing the rolling resistance.^[46]
- Dimensions - rolling resistance in tires is related to the flex of sidewalls and the contact area of the tire^[47] For example, at the same pressure, wider bicycle tires flex less in sidewalls as they roll and thus have lower rolling resistance (although higher air resistance).^[47]
- Extent of inflation - Lower pressure in tires results in more flexing of sidewalls and higher rolling resistance.^[47] This energy conversion in the sidewalls increases resistance and can also lead to overheating and may have played a part in the infamous Ford Explorer rollover accidents.
- Over inflating tires (such a bicycle tires) may not lower the overall rolling resistance as the tire may skip and hop over the road surface. Traction is sacrificed, and overall rolling friction may not be reduced as the wheel rotational speed changes and slippage increases.
- Sidewall deflection is not a direct measurement of rolling friction. A high quality tire with a high quality (and supple) casing will allow for more flex per energy loss than a cheap tire with a stiff sidewall. Again, on a bicycle, a quality tire with a supple casing will still roll easier than a cheap tire with a stiff casing. Similarly, as noted by Goodyear truck tires, a tire with a "fuel saving" casing will benefit the fuel economy through many tread lives (i.e. retreading), while a tire with a "fuel saving" tread design will only benefit until the tread wears down.
- In tires, tread thickness and shape has much to do with rolling resistance. The thicker and more contoured the tread, the higher the rolling resistance^[47] Thus, the "fastest" bicycle tires have very little tread and heavy duty trucks get the best fuel economy as the tire tread wears out.
- Diameter effects seem to be negligible, provided the pavement is hard and the range of diameters is limited. See section Depends on diameter
- Virtually all world speed records have been set on relatively narrow wheels, probably because of their aerodynamic advantage at high speed, which is much less important at normal speeds.
- Temperature: with both solid and pneumatic tires, rolling resistance has been found to decrease as temperature increases (within a range of temperatures: i.e. there is an upper limit to this

effect)^{[48][49]} For a rise in temperature from 30 °C to 70 °C the rolling resistance decreased by 20-25%.^[50] It is claimed that racers heat their tire before racing.

Railroads: Components of rolling resistance

In a broad sense rolling resistance can be defined as the sum of components^[51]):

1. Wheel bearing torque losses.
2. Pure rolling resistance.
3. Sliding of the wheel on the rail.
4. Loss of energy to the roadbed (and earth).
5. Loss of energy to oscillation of railway rolling stock.

Wheel bearing torque losses can be measured as a rolling resistance at the wheel rim, C_{rr} . Railroads normally use roller bearings which are either cylindrical (Russia)^[52] or tapered (United States).^[53] The specific rolling resistance in Russian bearings varies with both wheel loading and speed.^[54] Wheel bearing rolling resistance is lowest with high axle loads and intermediate speeds of 60–80 km/h with a C_{rr} of 0.00013 (axle load of 21 tonnes). For empty freight cars with axle loads of 5.5 tonnes, C_{rr} goes up to 0.00020 at 60 km/h but at a low speed of 20 km/h it increases to 0.00024 and at a high speed (for freight trains) of 120 km/h it is 0.00028. The C_{rr} obtained above is added to the C_{rr} of the other components to obtain the total C_{rr} for the wheels.

Comparing rolling resistance of highway vehicles and trains

Rolling resistance of the steel wheels on steel rail of a train is far less than the rubber tires wheels of an automobile or truck, but trains are much heavier per passenger or per net ton of freight. For 1975, Amtrak passenger trains weighed a little over 7 tonnes per passenger^[55] while automobiles weighed only a little over one ton per passenger. This means that much of the energy savings of the lower rolling resistance of a train is lost to its greater weight. However, CSX ran an advertisement campaign in 2013 claiming that their freight trains move "a ton of freight 436 miles on a gallon of fuel", whereas some sources claim trucks move a ton of freight about 130 miles per gallon of fuel, indicating trains are more efficient overall.

See also

- Coefficient of friction
- Low-rolling resistance tires
- Maglev (Magnetic Levitation, the elimination of rolling and thus rolling resistance)
- Rolling element bearing

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4. ^a A handbook for the rolling resistance of pneumatic tires Clark, Samuel Kelly; Dodge, Richard N. 1979 (<http://deepblue.lib.umich.edu/handle/2027.42/4274>)
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6. ^a Tyres-Online: The Benefits of Silica in Tyre Design (<http://www.tyrepriceadvisor.co.uk/en/top-tyre-brands/falken-tyres>)
7. ^a Астахов, p.85
8. ^a Деев, p. 79. Hay, p.68
9. ^a Астахов, Chapt. IV, p. 73+; Деев, Sect. 5.2 p. 78+; Hay, Chapt. 6 "Train Resistance" p. 67+
10. ^a Астахов, Fig. 4.14, p. 107
11. ^a If one were to assume that the resistance coefficients (Crr) for motor vehicles were the same as for trains, then for trains the neglected resistances taken together have a Crr of about 0.0004 (see Астахов, Fig. 4.14, p.107 at 20km/hr and assume a total Crr =0.0010 based on Fig. 3.8, p.50 (plain bearings) and adjust for roller bearings based on a delta Crr of 0.00035 as read from Figs. 4.2 and 4.4 on pp. 74, 76). Compare this Crr of 0.0004 to motor vehicle tire Crr's of at least 10 times higher per "Rolling resistance coefficient examples" in this article
12. ^a kgf/tonne is used by Астахов throughout his book
13. ^a Деев uses N/T notation. See pp. 78-84.
14. ^a http://www.biketechreview.com/tires/AFM_tire_crr.htm
15. ^a Hersey, equation (2), p. 83
16. ^a Астахов, p. 81.
17. ^a Hay, Fig. 6-2 p.72(worst case shown of 0.0036 not used since it is likely erroneous)
18. ^a Астахов, Figs. 3.8, 3.9, 3.11, pp. 50-55; Figs. 2.3, 2.4 pp. 35-36. (Worst case is 0.0024 for an axle load of 5.95 tonnes with obsolete plain (friction --not roller) bearings
19. ^a Астахов, Fig. 2.1, p.22
20. ^a "Coefficients of Friction in Bearing" (<http://www.tribology-abc.com/abc/cof.htm>). *Coefficients of Friction*. Retrieved 7 February 2012.
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22. ^a http://www.biketechreview.com/tires/images/AFM_tire_testing_rev8.pdf
23. ^{a b} Roche, Schinkel, Storey, Humphris & Guelden, "Speed of Light." ISBN 0-7334-1527-X
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25. ^a Green Seal 2003 Report (http://greenseal.org/resources/reports/CGR_tire_rollingresistance.pdf)

26. ^{a b} Gillespie ISBN 1-56091-199-9 p117
27. ^{a b c} Baker, Ira O., "Treatise on roads and pavements". New York, John Wiley, 1914. Stagecoach: Table 7, p. 28. Diameter: pp. 22-23. This book reports a few hundred values of rolling resistance for various animal-powered vehicles under various condition, mostly from 19th century data.
28. ^a Hersey, subsection: "End of dark ages", p.261
29. ^a Hersey, subsection: "Static rolling friction", p.266.
30. ^{a b} Williams, 1994, Ch. "Rolling contacts", eq. 11.1, p. 409.
31. ^a Hersey, subsection: "Coulomb on wooden cylinders", p. 260
32. ^a U.S. National Bureau of Standards, Fig. 1.13
33. ^a Some think that smaller tire wheels, all else being equal, tend to have higher rolling resistance than larger wheels. In some laboratory tests, however, such as Greenspeed test results (accessdate = 2007-10-27) (<http://www.legslarry.beerdrinkers.co.uk/tech/GS.htm>), smaller wheels appeared to have similar or lower losses than large wheels, but these tests were done rolling the wheels against a small-diameter drum, which would theoretically remove the advantage of large-diameter wheels, thus making the tests irrelevant for resolving this issue. Another counter example to the claim of smaller wheels having higher rolling resistance can be found in the area of ultimate speed soap box derby racing. In this race, the speeds have increased as wheel diameters have decreased by up to 50%. This might suggest that rolling resistance may not be increasing significantly with smaller diameter within a practical range, if any other of the many variables involved have been controlled for. See talk page.
34. ^a Gérard-Philippe Zéhil, Henri P. Gavin, Three-dimensional boundary element formulation of an incompressible viscoelastic layer of finite thickness applied to the rolling resistance of a rigid sphere, International Journal of Solids and Structures, Volume 50, Issue 6, 15 March 2013, Pages 833-842, ISSN 0020-7683, 10.1016/j.ijsolstr.2012.11.020.(journal article (<http://www.sciencedirect.com/science/article/pii/S002076831200491X>);author's page (1) (http://people.duke.edu/~gpz/rolling_resistance.html);author's page (2) (<http://people.duke.edu/~hpgavin/rollingresistance/>))
35. ^a Gérard-Philippe Zéhil, Henri P. Gavin, Simple algorithms for solving steady-state frictional rolling contact problems in two and three dimensions, International Journal of Solids and Structures, Volume 50, Issue 6, 15 March 2013, Pages 843-852, ISSN 0020-7683, 10.1016/j.ijsolstr.2012.11.021.(journal article (<http://www.sciencedirect.com/science/article/pii/S0020768312004921>);author's page (1) (http://people.duke.edu/~gpz/rolling_resistance.html);author's page (2) (<http://people.duke.edu/~hpgavin/rollingresistance/>))
36. ^a Gérard-Philippe Zéhil, Henri P. Gavin, Simplified approaches to viscoelastic rolling resistance, International Journal of Solids and Structures, Volume 50, Issue 6, 15 March 2013, Pages 853-862, ISSN 0020-7683, 10.1016/j.ijsolstr.2012.09.025.(journal article (<http://www.sciencedirect.com/science/article/pii/S002076831200409X>);author's page (1) (http://people.duke.edu/~gpz/rolling_resistance.html);author's page (2) (<http://people.duke.edu/~hpgavin/rollingresistance/>))
37. ^{a b} Roberts, Fig. 17: "Effect of torque transmission on rolling resistance", p. 71
38. ^a Деев, p.30 including eq. (2.7) and Fig. 2.3
39. ^a Астахов, Figs. 3.8, 3.9, 3.11, pp. 50-55. Hay, Fig. 60-2, p. 72 shows the same phenomena but has higher values for Crr and not reported here since the railroads in 2011 [1] (<http://www.istc.illinois.edu/about/SeminarPresentations/20091118.pdf>). were claiming about the same value as Астахов

40. ^ Hersey, Table 6., p. 267
41. ^ Per this assumption, $F = kN^{0.5}$ where F is the rolling resistance force and N is the normal load force on the wheel due to vehicle weight, and k is a constant. It can be readily shown by differentiation of F with respect to N using this rule that $\frac{dN}{N} = 2\frac{dF}{F}$
42. ^ Roberts, pp. 60-61.
43. ^ [2] (<http://www.springerlink.com/content/x1707075n815g604/>) C. Michael Hogan, *Analysis of Highway Noise*, Journal of Soil, Air and Water Pollution, Springer Verlag Publishers, Netherlands, Volume 2, Number 3 / September, 1973
44. ^ Gwidon W. Stachowiak, Andrew William Batchelor, *Engineering Tribology*, Elsevier Publisher, 750 pages (2000) ISBN 0-7506-7304-4
45. ^ <http://144.206.159.178/ft/200/607426/12614863.pdf>
46. ^ http://www.rubberchemtechnol.org/resource/1/rctea4/v3/i1/p19_s1?isAuthorized=no
47. ^ *a b c d* "Schwalbe Tires: Rolling Resistance" (http://www.schwalbetires.com/tech_info/rolling_resistance).
48. ^ The Recumbent Bicycle and Human Powered Vehicle Information Center (<http://www.recumbents.com/mars/pages/proj/tetz/other/Crr.html>)
49. ^ U.S National Bureau of Standards p.? and Williams p.?
50. ^ Roberts, "Effect of temperature", p.59
51. ^ Астахов, p. 74, Although Астахов list these components, he doesn't give the sum a name.
52. ^ Шадур. Л. А. (editor). Вагоны (**Russian**)(Railway cars). Москва, Транспорт, 1980. pp. 122 and figs. VI.1 p. 123 VI.2 p. 125
53. ^ Association of American Railroads, Mechanical Division "Car and Locomotive Encyclopedia", New York, Simmons-Boardman, 1974. Section 14: "Axle journals and bearings". Almost all of the ads in this section are for the tapered type of bearing.
54. ^ Астахов, Fig 4.2, p. 76
55. ^ Statistics of railroads of class I in the United States, Years 1965 to 1975: Statistical summary. Washington DC, Association of American Railroads, Economics and Finance Dept. See table for Amtrak, p.16. To get the tons per passenger divide ton-miles (including locomotives) by passenger-miles. To get tons-gross/tons-net, divide gross ton-mi (including locomotives) (in the "operating statistics" table by the revenue ton-miles (from the "Freight traffic" table)
- Астахов П.Н. (**Russian**) "Сопротивление движению железнодорожного подвижного состава" (Resistance to motion of railway rolling stock) Труды ЦНИИ МПС (ISSN 0372-3305). Выпуск 311 (Vol. 311). - Москва: Транспорт, 1966. – 178 pp. perm. record at UC Berkeley (<http://oskicat.berkeley.edu/record=b12003170~S1>) (In 2012, full text was on the Internet but the U.S. was blocked)
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(<http://webphysics.davidson.edu/faculty/dmb/PY430/Friction/rolling.html>)

- temperature vs rolling resistance
(<http://www.recumbents.com/mars/pages/proj/tetz/other/Crr.html>)
- Simple roll-down test to measure Crr in cars and bikes
(<http://physics.technion.ac.il/~rutman/car/Roll-down%20test.pdf>)

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